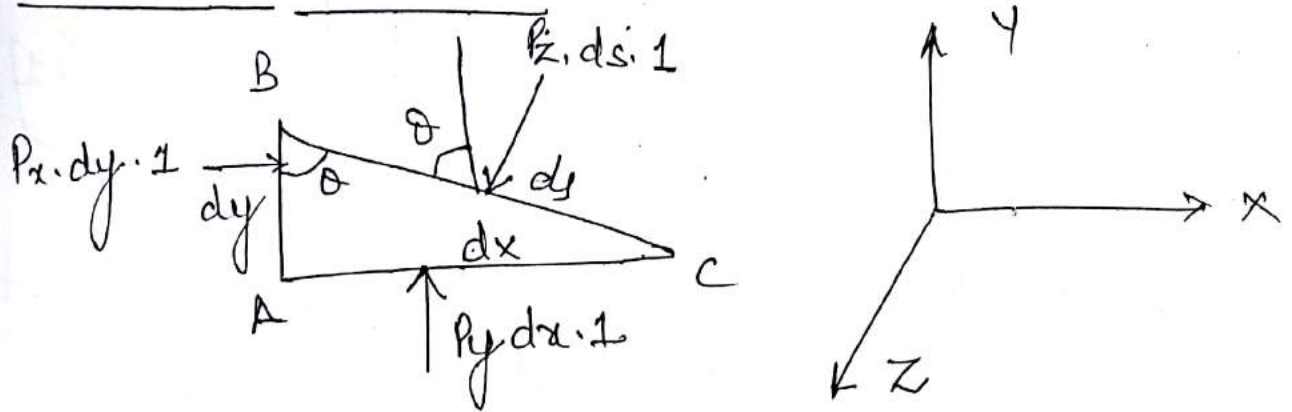


Pascal's Law: →



"It states that the pr. or pr. intensity at a point in a static fluid is equal in all directions!"

The fluid element is of very small dimensions (i.e. dx , dy and ds).

Let the width of the element perpendicular to the plane of paper be unity and P_x , P_y and P_z are the pr. intensity acting on the faces AB, AC and BC. Let $\angle ABC = \theta$

then the force acting on the element are -

1. In forces normal to the surfaces
2. Weight of element in the vertical direction

$$\begin{aligned} \text{Force on the face AB} &= P_x \times \text{Area of face} \\ &= P_x \times dy \times 1 \end{aligned}$$

$$\text{Similarly force on AC} = P_y \times dx \times 1$$

$$\text{" " BC} = P_z \times ds \times 1$$

consider

$$\begin{aligned} \text{Weight of Element} &= \text{Mass of element} \times g \quad (2) \\ &= (\text{Volume} \times \rho) \times g = \left(\frac{1}{2} AB \times AC \times 1 \right) \times \rho \times g \end{aligned}$$

Resolving the forces in x-direction —

$$P_x \times dy \times 1 - P_z \cdot ds \cdot 1 \sin(90^\circ - \theta) = 0$$

$$P_x \times dy \times 1 - P_z ds \cos \theta = 0$$

But $ds \cos \theta = AB = dy$

$$P_x \times dy \times 1 - P_z dy \times 1 = 0$$

$$\boxed{P_x = P_z} \quad \text{--- (i)}$$

Similarly resolving the force in y-directions

$$P_y \times dx \times 1 - P_z \times ds \times 1 \cos(90^\circ - \theta)$$

$$- \text{weight of element} = 0$$

$$P_y \times dx - P_z ds \sin \theta - \frac{1}{2} dx \times dy \times \rho \times g = 0$$

$$P_y \times dx - P_z dx - \text{---} = 0$$

since element is very small and hence is negligible

$$\boxed{P_y = P_z} \quad \text{--- (ii)}$$

from eq. (i) and (ii)

$$\boxed{P_x = P_y = P_z} \quad \text{--- (iii)}$$

∴ at any point same in all directions,

Applications! →

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ME-Dept

(1) - Hydraulic Press

(2) - In Braking System.

Pressure Variation in A. Fluid, At Rest! →

Hydrostatic Law! →

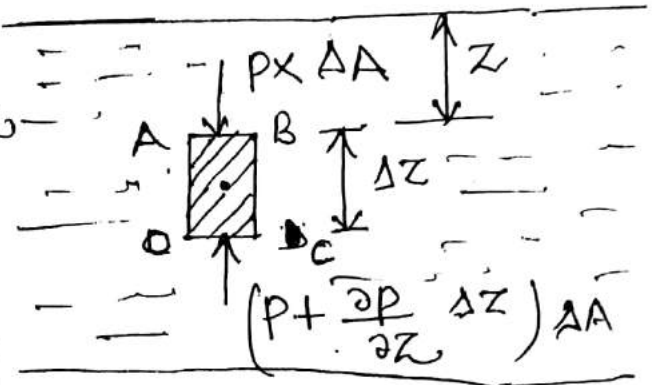
Hydrostatic law states

that the rate of increase of pressure in a

vertically downward

directions must be equal

to the specific weight of the fluid at that point.



Consider a small fluid element -

Let ΔA = cross sectional Area of element

Δz = Height of fluid element

P = Pr. on face AB

z = Distance of fluid element from free surface.

The forces acting on the fluid element are: -

1. Pr. force on AB = $P \times \Delta A$ ↓

2. Pr. force on CD = $(P + \frac{\partial P}{\partial z} \Delta z) \times \Delta A$ ↑

insider

3. Weight of fluid element = $\rho \times g \times \text{volume}$ (4)
 $= \rho \times g \times (\Delta A \times \Delta z)$

4. Pr. forces on surfaces BC and AD are equal and opposite.

For eq. of fluid element: \rightarrow

$$P \Delta A - \left(P + \frac{\partial P}{\partial z} \Delta z \right) \Delta A + \rho g \times (\Delta z \times \Delta A) = 0$$

$$-\frac{\partial P}{\partial z} \Delta z \Delta A + \rho g \times \Delta A \Delta z = 0$$

OR $\frac{\partial P}{\partial z} \Delta z \Delta A = \rho g \times \Delta A \Delta z$

$$\frac{\partial P}{\partial z} = \rho g \quad \text{--- (i)}$$

By integrating the above eq -
 $\int dp = \int \rho g z$

$$\boxed{P = \rho g z} \quad \text{--- (ii)}$$

$z = \text{pressure head}$